

Energy savings in data centers: A framework for modelling and control of servers' cooling

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Definition: Data center

From Wikipedia¹:

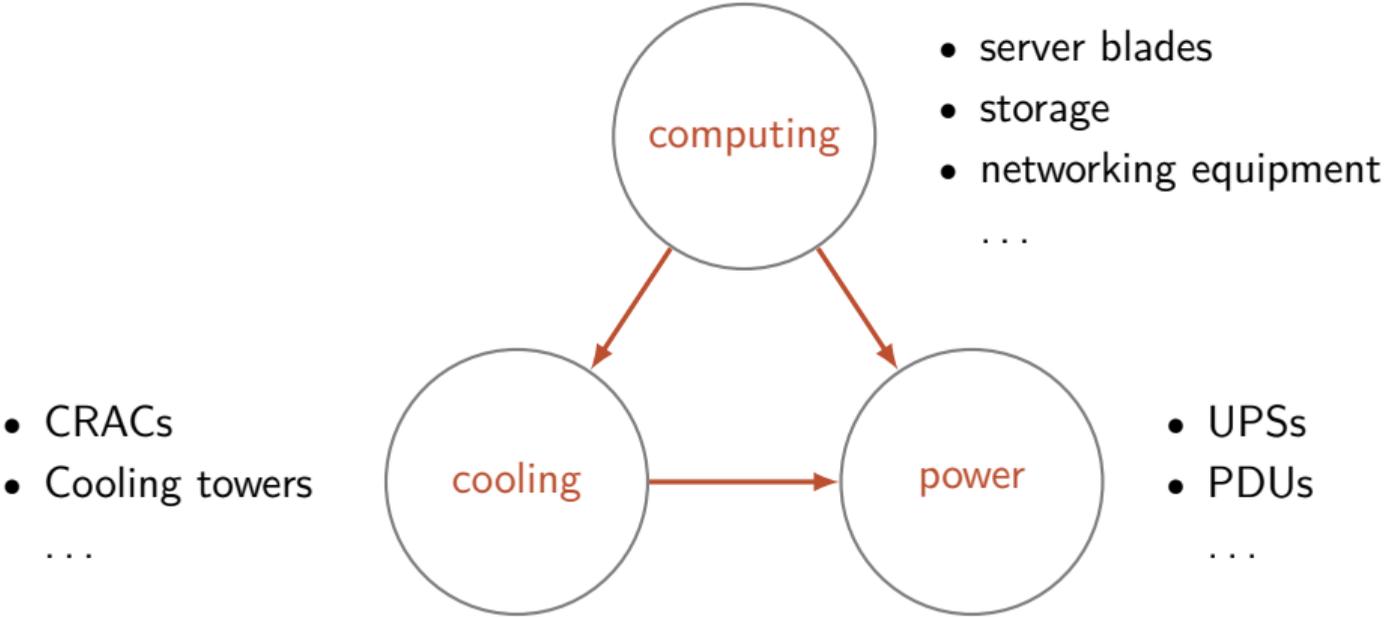
- [...] a **facility used to house computer systems** and associated components
- the main purpose [...] is **running the IT systems applications** [...] of the organization
- [...] large data centers are **industrial scale operations** using as much electricity as a small town

¹[wikipedia.org/wiki/Data_center](https://en.wikipedia.org/wiki/Data_center)





Structures within a data center and their logical connections



Footprint

EU-28: (year 2013)

- Industry market: 18.85 billion €
- Electrical power consumption: 103.4 GWh
(~ 3% of electrical power production)
- 38.6 metric tonnes of CO_2 emissions (347g/kWh)

Is there a problem to solve?

- i) Electronics convert virtually all power into heat
- ii) Cooling accounts for up to **40%** of power budget
- iii) Computing's power consumption dependent on thermodynamical state
- iv) DCs rarely operate at peak computing capacity²

Problem: Efficiency (static, nominal, cooling policies waste power)

²Often desired since it is paramount to maintain Quality of Service

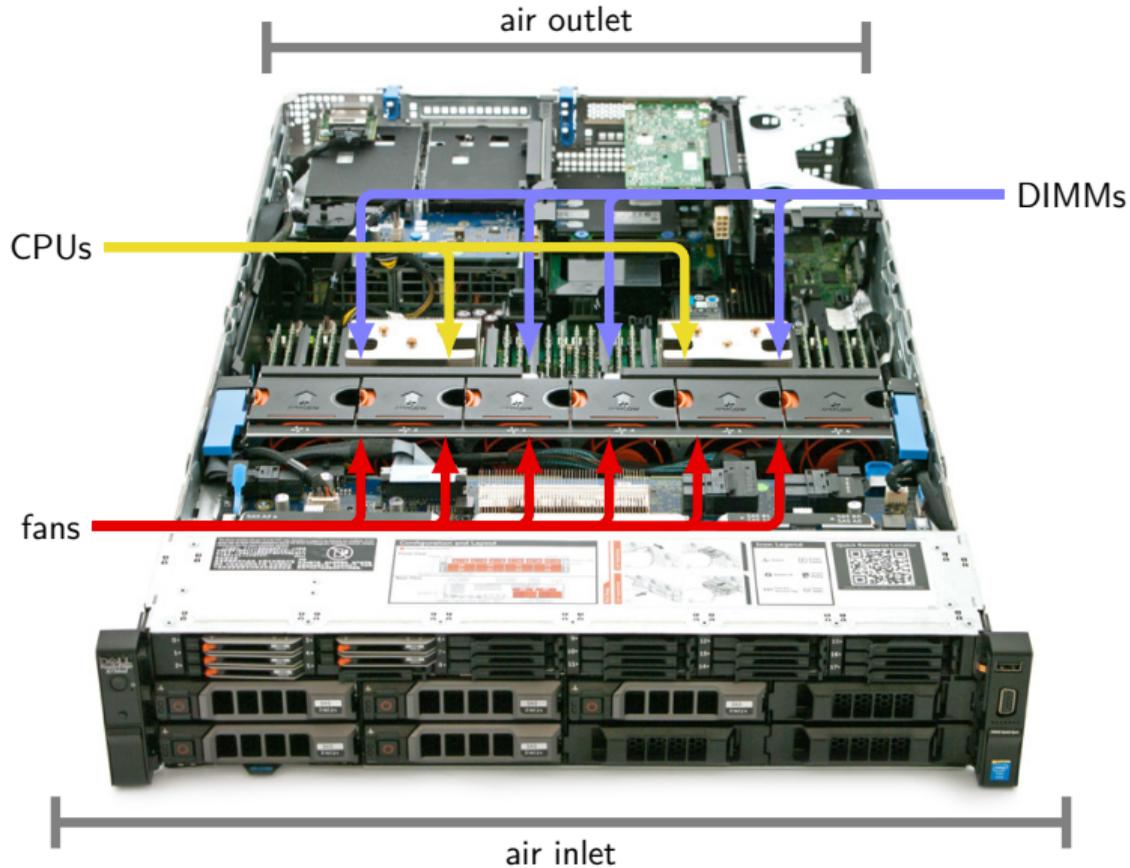
Control-oriented solution

Measure & predict the thermodynamical state
to reduce overprovision of the cooling resources

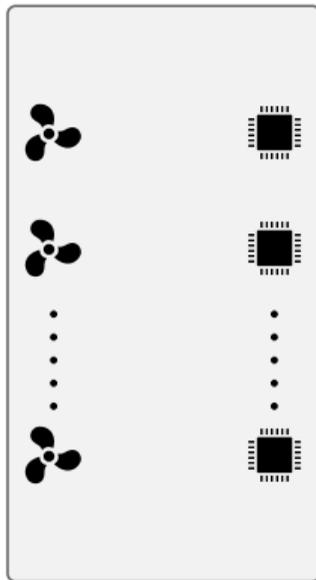
Potential control levels

- i) single server level (*focus of this paper*)
- ii) rack level
- iii) room level

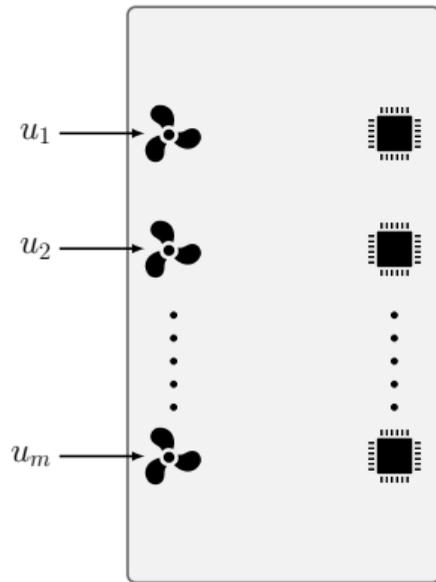
Example of “atomic unit”: air-cooled blade



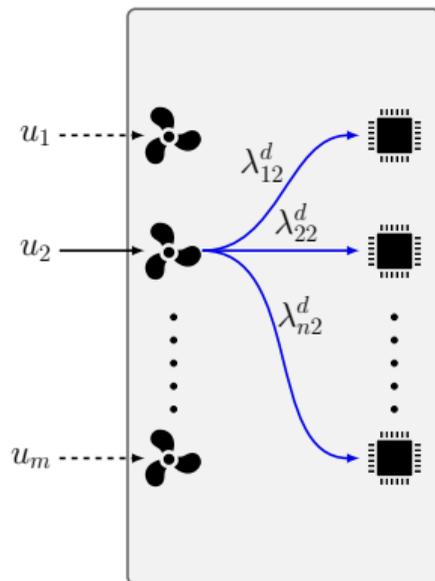
Modeling air-cooled blades, intuitions



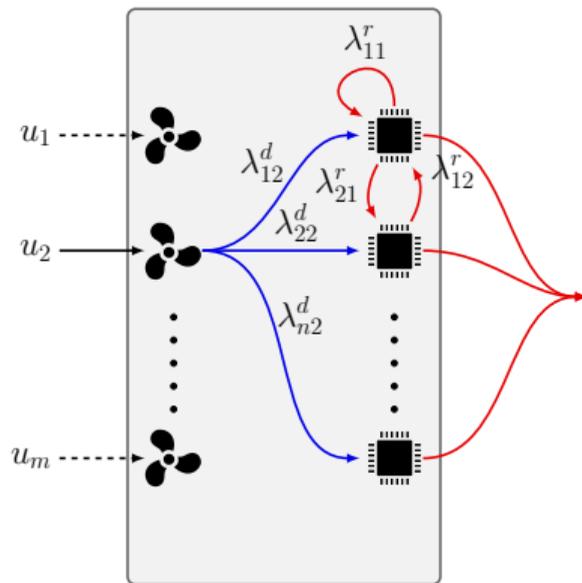
Modeling air-cooled blades, intuitions



Modeling air-cooled blades, intuitions



Modeling air-cooled blades, intuitions



Modeling air-cooled blades, in math

Notation

states:

- $\mathbf{x}^c = [x_1^c \ \dots \ x_n^c]^T$ =: temperatures of IT components
- $\mathbf{x}^f = [x_1^f \ \dots \ x_n^f]^T$ =: temperatures of air flows through IT components

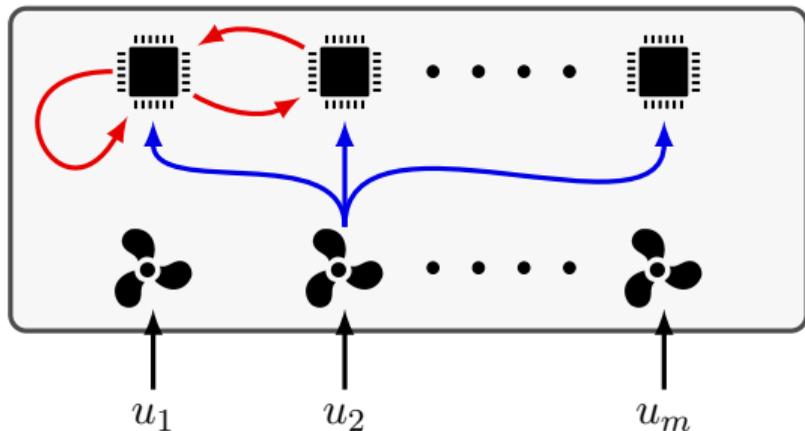
exogenous inputs:

- $\mathbf{p} = [p_1 \ \dots \ p_n]^T$ =: electrical power dissipated by IT components
- x^i =: temperature of air inlet

manipulable inputs:

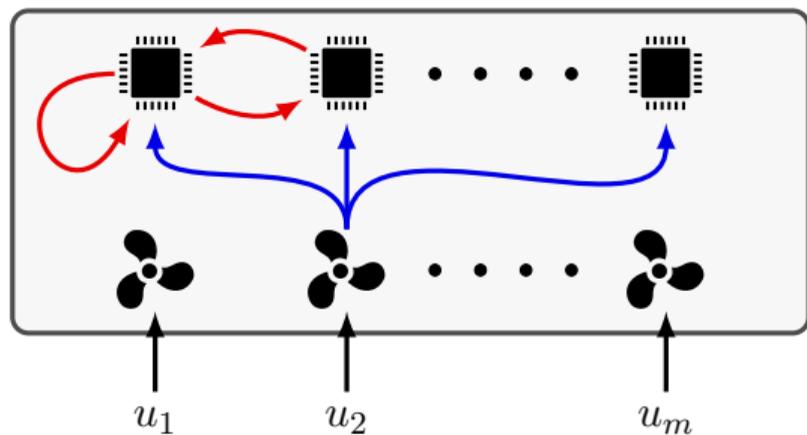
- $\mathbf{u} = [u_1 \ \dots \ u_m]^T$ =: air mass flows produced by fans
(determining the air flows through the IT components $\mathbf{f} = [f_1 \ \dots \ f_n]^T$)

Ingredients to be modeled



- i) mass of the air flows f
- ii) temperature of the air flows x^f
- iii) temperature of the IT components x^c

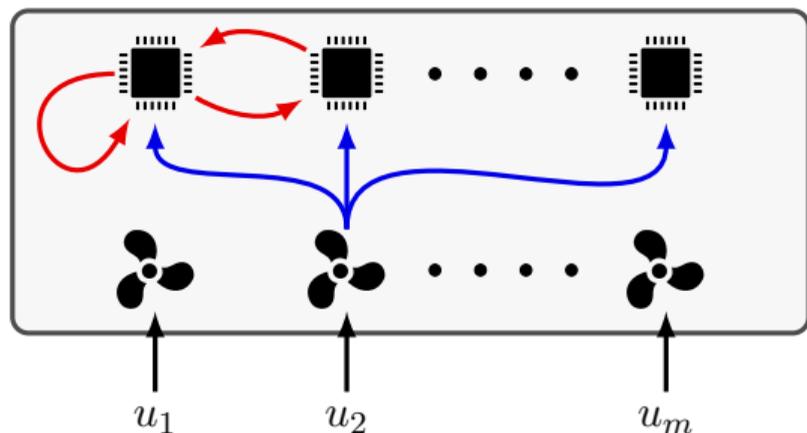
Model of the mass of the air flows



$$\mathbf{f} = \Lambda^d[\mathbf{u}] + \Lambda^r[\mathbf{f}] \quad (1)$$

$$\Lambda[\mathbf{u}] = \sum_{i=1}^{(m+d)} a_i \mathbf{u}^{\alpha_i} \quad (2)$$

Model of the mass of the air flows



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Generalizes:

- i) models without *warm* recirculation flows ($\Lambda^r = 0$)
- ii) models preserving total mass-flows (Λ^d and Λ^r row-stochastic matrices)

Model of the temperature of the air flows

Assumptions:

- i) perfect flow mixing
- ii) heat energy conservation

$$x_j^f = \frac{\Lambda_{(j)}^d[\mathbf{u}] \cdot x^i + \sum_{h=1}^n \Lambda_{(j,h)}^r[\mathbf{f}] \cdot x_h^c}{f_j} \quad (3)$$

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corresponds to averaging

Model of the temperature of the IT components

$$\begin{aligned} \dot{x}_j^c &= \underbrace{-h_j f_j(\mathbf{u}) (x_j^c - x_j^f(\mathbf{u}, \mathbf{x}^c, x^i))}_{\text{convection}} \\ &+ \underbrace{\begin{bmatrix} R_{(j)} & \rho_j \end{bmatrix} \begin{bmatrix} \mathbf{x}^c \\ x^i \end{bmatrix}}_{\text{conduction}} \\ &+ \underbrace{b_j p_j}_{\text{electrical power}} \end{aligned} \quad (4)$$

Model of the temperature of the IT components

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Euler forward discretization:

$$x_j^c(k+1) = \Psi_j^\Delta(x_j^c(k), \mathbf{u}, \mathbf{p}, x^i)\tag{5}$$

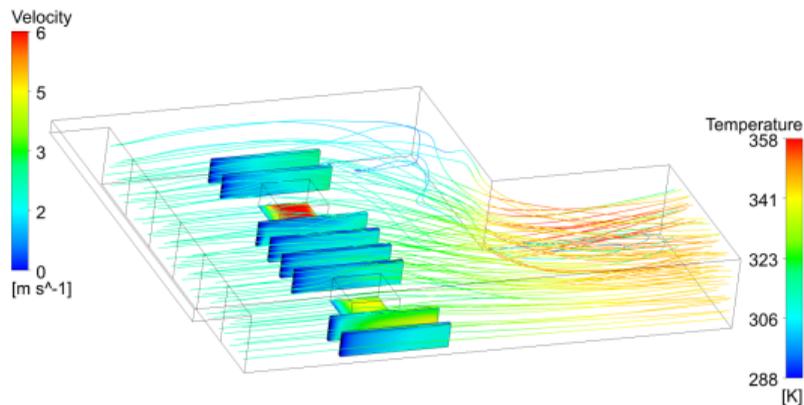
Minimum cost fan control

$$\begin{aligned} & \min_{\mathbf{u}(0), \dots, \mathbf{u}(H-1)} \sum_{t=0}^{H-1} \sum_{h=1}^m (u_h(t))^3 \\ & \text{subject to (for } 0 \leq t \leq H-1, 1 \leq j \leq n): \\ & \quad \mathbf{x}^c(0) = \mathbf{x}_0^c \\ & \quad \mathbf{u}_{\min} \leq \mathbf{u}(t) \leq \mathbf{u}_{\max} \\ & \quad \mathbf{x}^c(t+1) \leq \mathbf{x}_{\max}^c \\ & \quad x_j^c(t+1) = \Psi_j^\Delta(x_j^c(t), \mathbf{u}(t), \mathbf{p}(t), x^i) \end{aligned} \tag{6}$$

numerical simulations

- i) how accurate is the polynomial flows model?
- ii) how conservative is the control strategy?

Validation of the model using CFD

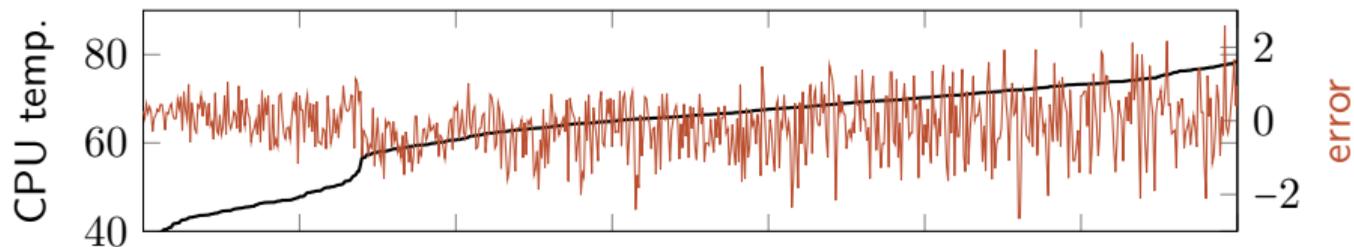


Algorithm:

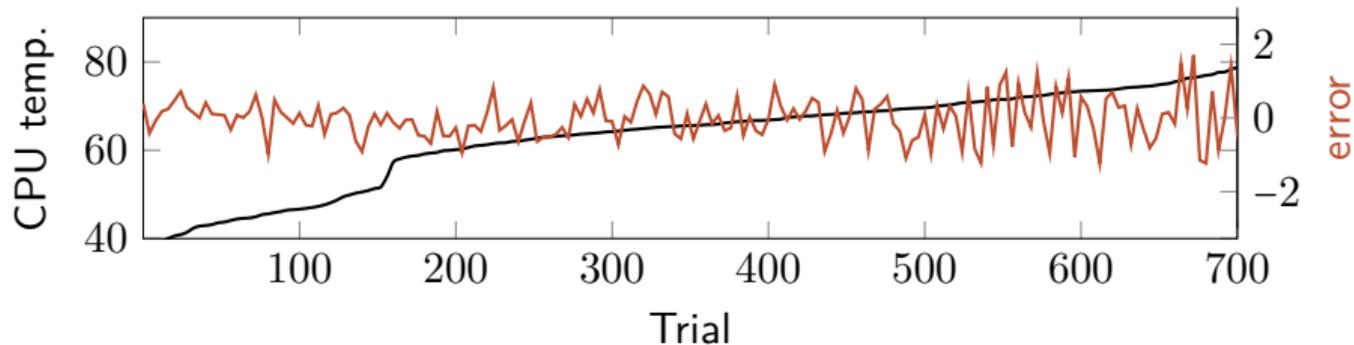
- 1 set boundary conditions (i.e., $\mathbf{p}, \mathbf{u}, x^i$)
- 2 use CFD model as a virtual plant
- 3 estimate the parameters using RLS

Validation of the model using CFD

$$\deg(\Lambda^d) = 2, \deg(\Lambda^r) = 1$$



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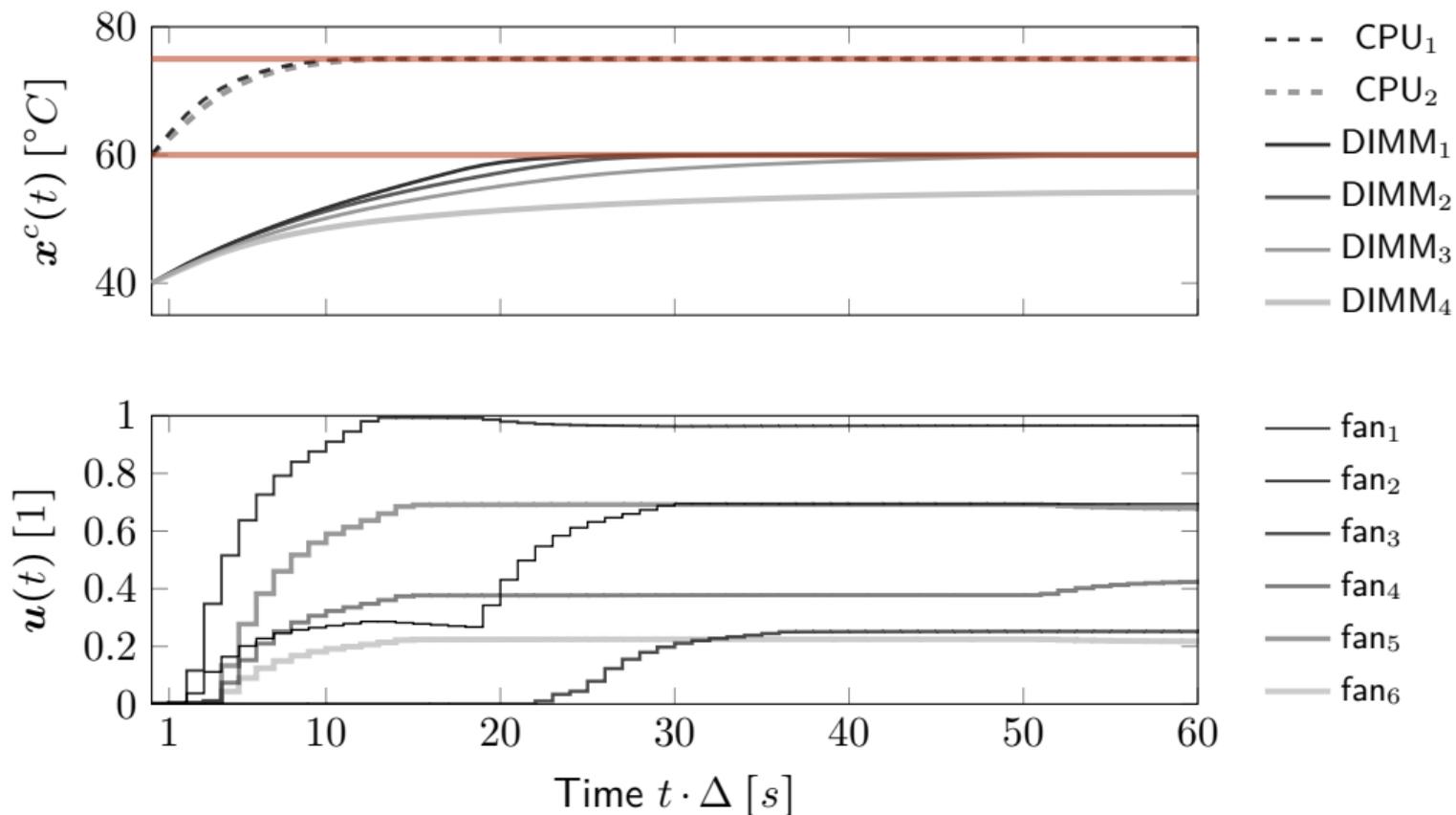


Validation of the control strategy

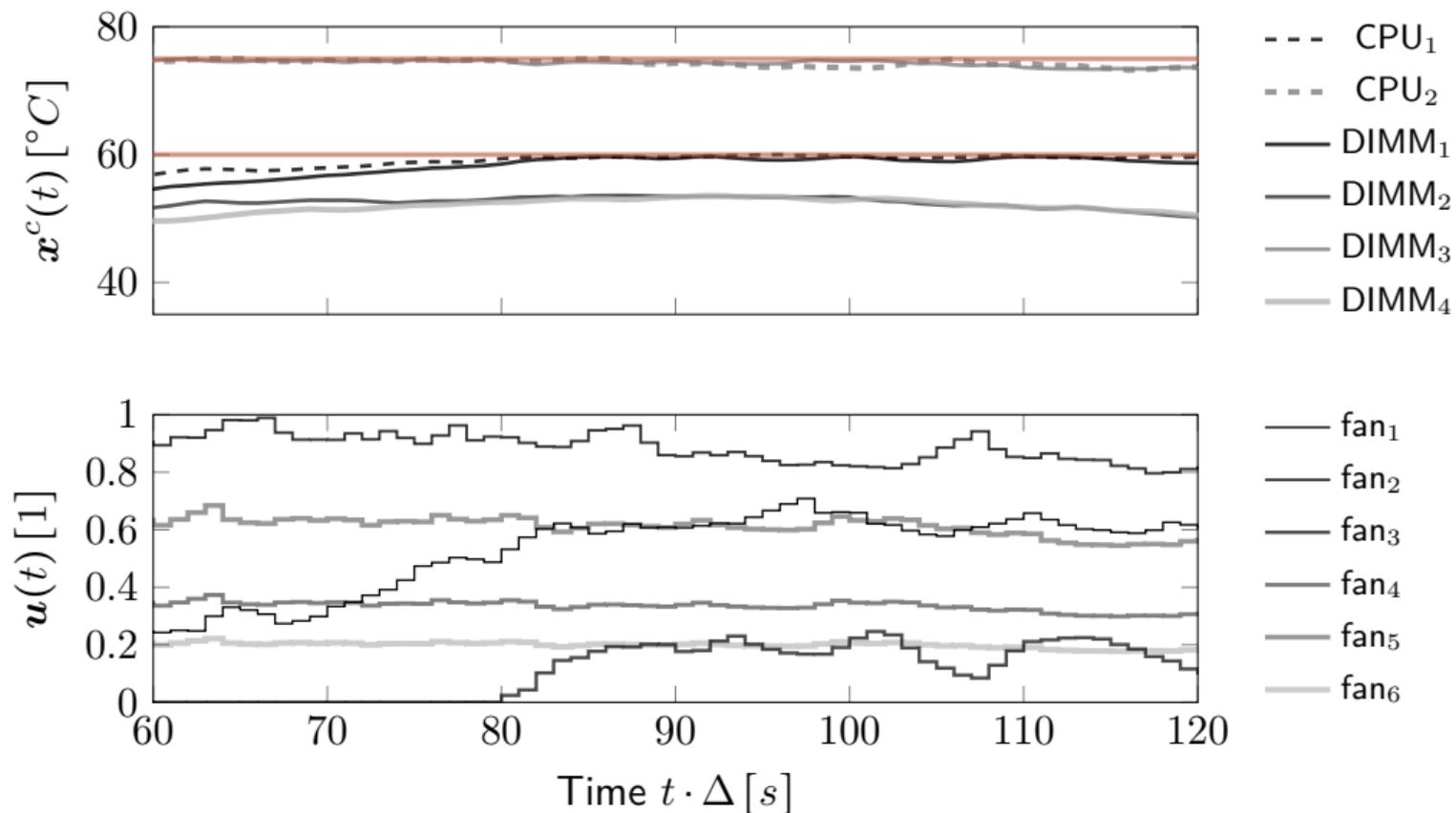
Experiment 1: $\mathbf{p}(t) = \mathbf{p}_{\max}$

Experiment 2: $\mathbf{p}(t)$ = stochastic process with seasonal trends (*medium-high loads*)

Validation of the control strategy - Experiment 1



Validation of the control strategy - Experiment 2



Conclusions

- i) promising ability at capturing complex convective cooling effects
- ii) functional structure open to other cooling applications
- iii) can be identified starting from CFD models

Conclusions and future works

- i) promising ability at capturing complex convective cooling effects
- ii) functional structure open to other cooling applications
- iii) can be identified starting from CFD models

- i) validate on real plants
- ii) generalize to “datacenter room control”
- iii) generalize to generic thermal networks

RISE SICS North ICE

- 3000 - 4000 servers (2MW)
- 160 m² lab
- biogas back up generators
- connections with the urban district heating network
- Generality, Flexibility and Expandability

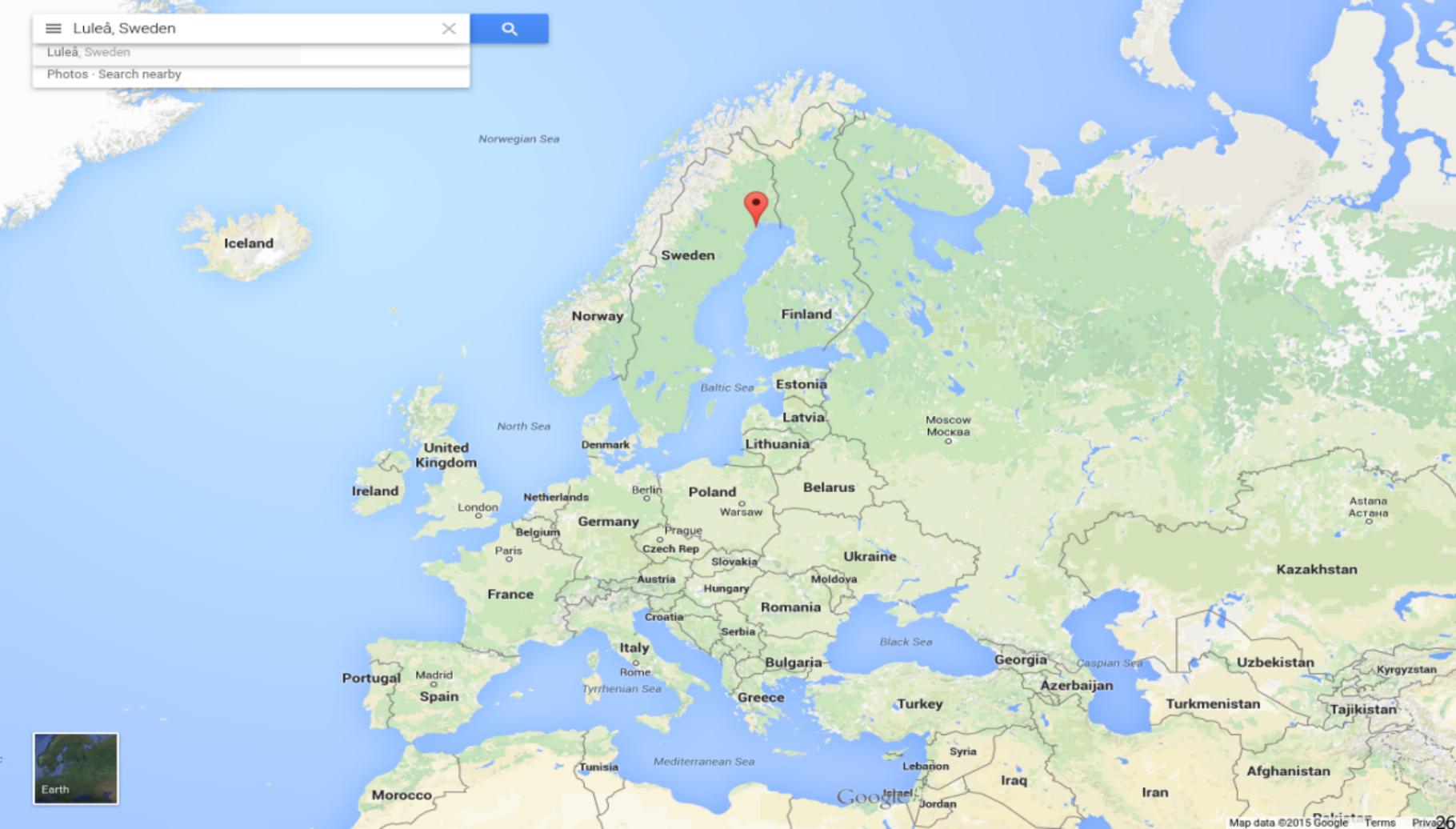
Experiment-as-a-Service

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